

T.Z. Farge

Electromechanical
Engineering Department,
University of Technology
Baghdad, Iraq
drtalebzf@yahoo.com

A.J. Owaid

Electromechanical
Engineering Department,
University of Technology
Baghdad, Iraq
aljabbar.2007@yahoo.com

M.A. Qasim

Electromechanical
Engineering Department,
University of Technology
Baghdad, Iraq
blueskyeng24@gmail.com

Received on: 29/09/2016

Accepted on: 16/03/2017

The Effect of Speed Smart Control System SSCS on the Performance of Hydropower System

*Abstract-*In this work, the speed smart control system is designed and implemented to improve and enhance the performance of hydropower system, where Arduino Uno R3 microcontroller is used for this propose. The speed smart control system is used to control the volume flow rate of water with respect to the load applied to the Pelton turbine shaft at optimum range of speed. Using nozzle outlet diameter of 8.87 mm. A water pump is used to generate the volume flow rate and pressure head. The results show that the maximum reduction in the hydraulic power was observed at zero torque, where the percentage reduction in the hydraulic power was equal to 87.33% when using speed smart control system. Also the optimum torque for maximum brake power and efficiency of Pelton turbine system have been increased when using a speed smart control system, where the percentage increasing was about 28.15%. Comparing result with and without using smart control system shows the percentage increased in the brake power and efficiency of Pelton turbine system were 26.3% and 35% respectively at the optimum torque for maximum brake power and efficiency of Pelton turbine system. Time response was four seconds to achieve a steady state for the rotational speed of Pelton turbine.

Keywords- Pelton Turbine, Water Pump, Electrical Power, Arduino Uno

How to cite this article: T.Z. Farage A.J. Owaid and M.A. Qasim, "The Effect of Speed Smart Control System SSCS on the Performance of Hydropower System", *Engineering and Technology Journal*, Vol. 35, No. 6, pp. 602-608, 2017.

1. Introduction

The Electricity power considered as very important in the world and especially in the Iraq country due to higher temperature during the summer [1]. The hydropower plant is one of the important sources of the renewable energy of the worldwide to generate the electricity. The percentage-generated energy by a hydropower was equal to 86.31% of the total renewable energy resisted by the international energy agency at 2012 [2]. The Pelton turbine is one of the most important part of the hydropower plants, which a type is of impales water turbine. In the 1870, the Pelton turbine was invented by luster Allan Pelton [3]. Niranjana et al. [4] developed a method to control the speed of control an induction motor. They were used an open loop phase control by using the Arduino controller. They were controlled the speed of induction motor could be controlled by using Arduino to controlling the pulses. Paul [5] implemented a persistence of vision a design based on advance microprocessor (Arduino duemilanove). They used Arduino due Milan one board because of higher speed at operation, easy to used lower power motor, and low cost than others microcontroller. The results shown that the display was extremely attractive to look and give a sense of being a transparent display. Neerparaj

and Bijay [6] developed a closed loop control system to control the speed of DC motor. They were used ATmega168 Arduino microcontroller. The results show that the system outputs were graduate with that obtained from the theoretical results. The Pelton turbine is type of an impales turbine, which convert the potential energy into the kinetic energy. There many papers have been published in the last decades about experimental and numerical analysis and design of Pelton turbine [7-19] to improve the performance and development of Pelton turbine. In addition, there are many papers have been published to study numerically and experimentally the performance of the nozzle which used in the hydropower [20-28]. The objective of the present work is to investigate experimentally the effect of the speed smart control system on the performance of the hydropower system. The speed smart control system is used to improve and enhance the performance of hydropower system, where Arduino Uno R3 micro controller was used for this propose.

2. Theory

The discharge of water, the torque applied on the turbine shaft and water head are the main parameters that effect the performance of

hydropower system. The volume flow rate) discharge) of the water is using to calculated [29].

$$Q = \frac{V}{t}$$

(1)

The input hydropower applied to the Pelton turbine is evaluated as:

$$P_h = \rho g H Q$$

(2)

The following equation is used to find the torque on the wheel of Pelton turbine:

$$T = (F_1 - F_2)R$$

(3)

The power produced by the Pelton turbine (brake power) is:

$$P_b = T \times \omega$$

(4)

The efficiency produced by the hydropower system is determined as:

$$\eta = \frac{P_b}{P_h} \times 100\%$$

(5)

3. Experimental Work

A test rig of a Pelton turbine system was designed and implemented as shown in Figures 1 and 2 show the speed smart control system used, where the experimental works were carried on it with nozzle of outer diameter of (8.78) mm. The system consists of Speed Smart Control System SSCS, Pelton turbine with 24cup buckets of the tip diameter of (269.89) mm and hub diameter of (221.29)mm as shown in Figure 3, water pump, digital flow meter, tachometer, and tension scale gauge. A water pump was used to generate volume flow rate and the pressure head.

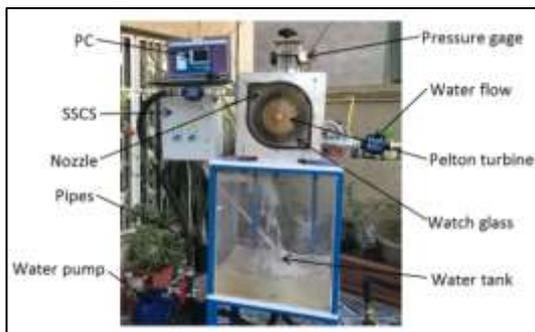


Figure 1: hydropower system

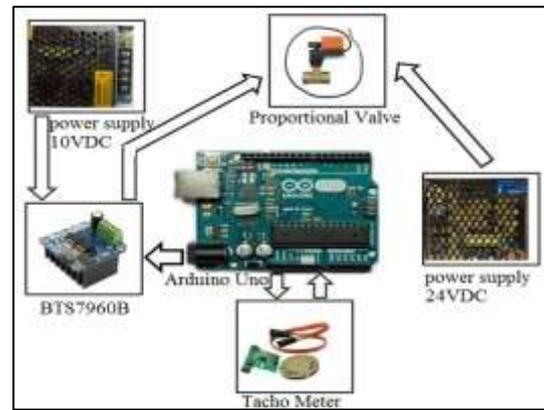


Figure2: Speed Smart Control System (SSCS)

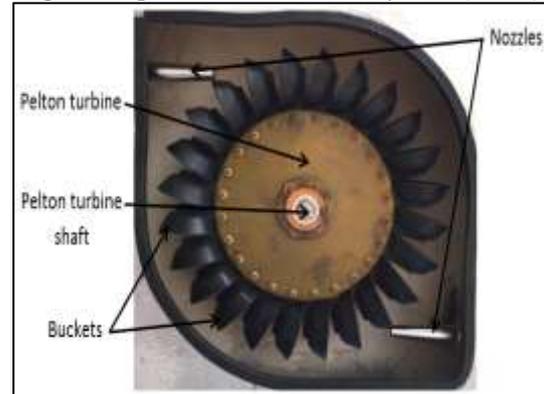


Figure 3: Pelton turbine

4. Results and Discussion

The experiment include a comparative study between Pelton turbine performance with and without a smart control system SSCS using nozzle with outlet diameter of (8.78mm). The controlled speeds of the Pelton turbine were between 530 and 585 (rpm), where the maximum values of brake power and efficiency were obtained at this range of speed. Fig.4. show the speed response of Pelton turbine with the torque equate to zero, where speed over shot from zero to 1400 rpm and then decreased towered the average value of the speed setting in about 4 seconds, when the control value rotated to partially closed the value and reduced the volume flow rate of the water. Fig.5. shows the speed response of Pelton turbine when increasing the torque where the speed curve was oscillated between the 40 and 50 seconds and then settled of the average value the controlling speed. Figures 6 to 13 show the comparative test results for Pelton turbine performance as a function of torque (load) applied on the turbine shaft with and without using smart speed control system. Figure 6 shows the volume flow rate was constant in the case of without using a control system, while in the case of using control system the volume flow rate has been variable, which has a lower value at zero torque and increase when the torque increases

until reach a certain value approximately equate to 1.29 N.m then the volume flow rate follow behavior without control system as shown in Figures 5 to 12. In the case of using a control system there is large save of amount of mass of water at lower applied load on the turbine shaft. The percentage decreased in the volume flow rate at zero torque is about 52.54% in case of using a control system comparing without using control system. Figure 7 shows the hydraulic power (input power) of the water equal constant value the in case of without using control system, because of constant volume flow rate and lead of water. While the hydraulic power is variable in the case of using control system due to the variable values volume flow rate of water head as shown in Figure 6 and Table 1. The maximum percentage reducing in the hydraulic power is approximately equal to 87.33% at zero torque. Figures 8 and 9 show the performance characteristic of Pelton turbine without and with using control system. The figures show the control system were improved and enhances performance for Pelton turbine by using the control system. Fig.8. shows that the maximum brake power was increased in the case of using control system, which enhances performance for the Pelton turbine due to the working conditions at the speed of optimum torque for maximum brake power and efficiency. Figure 9 shows the improvement in the efficiency of the Pelton turbine in the case of using control system when the torque less than 1.4 N.m. This is because of reduction in the hydraulic power due to their action of control system, where the maximum efficiency in the case of using a control system was about 57.7%, while in case of without using a control system was about of 37.5%. Figures 10 to 13 the relation of speed of Pelton turbine and the performance characteristic of Pelton turbine system without and with using control system, where the setting speed for controlling system where between 530 and 585 (rpm). These figures show there are three regions of variation of the curves with respect to the rotational speed of the Pelton turbine .the first region is of constant rotational speed at the optimum torque for maximum brake power and efficiency of Pelton turbine system. The second region is between the constant rotational speed and 200 rpm. While the third region is between 200 and zero rpm, which follow the performance characteristic of Pelton turbine system without using controlling system. In this case the Pelton turbine system because out of control. Fig.10. show the behavior of torque with respect to the rotational speed control system and turbine by using a control system. In the case

of without using control system the torque distribution is linearly with respect to the rotational speed (zero torque at maximum speed and maximum torque with zero rotational speed). While in the case of using control system, the torque was distributed along the constant rotational speed of Pelton turbine at the optimum torque for maximum brake power and efficiency in the first. While the torque distribution in the second and third regions become linearly as show in the Figure 10, any increase in the torque lead to the reduction in rotational speed of Pelton turbine. Also the figure show that an improvement and enhancement in the torque in the first and second regions, where the optimum torque for maximum brake power and efficiency was increased from 0.92675 N.m for without control to 1.29N.m (the percentage increasing in the torque was 28.15%). This improvement led to improve in the brake power and efficiency of Pelton turbine system as shown in Figures 11 and 12. Figure 13 shows the volume flow rate distribution for by using control system and without using control system with respect to rotational speed of Pelton turbine. In the case of without using control system the volume flow rate was constant and had a value of 5.9 l/min. This was indicated a large saving in the water consuming for the required torque. Where the percentage saving of water volume flow rate was 52.54% for using control system at zero torque. Figures 12 and 13 show the improvement and enhancement of the Pelton turbine system by using a smart speed control system especially at the first region of variation due to increase in the optimum torque and reduction in hydraulic power as explained previously. Where the percentage increasing in the brake power and efficiency of Pelton turbine were 26.31% and 35% respectively at optimum torque for the maximum brake power and efficiency of Pelton turbine system.

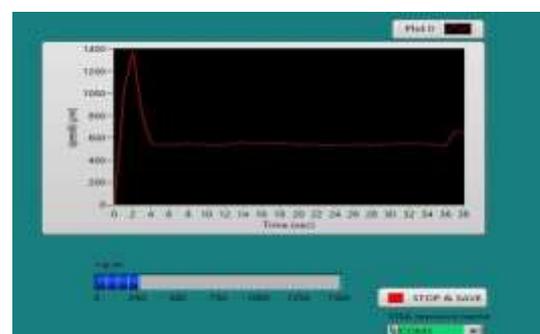


Figure4: Pelton turbine speed response by using speed smart control system with zero torque

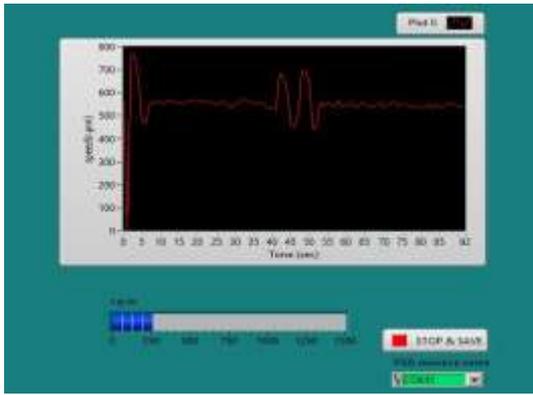


Figure 5: Pelton turbine speed response by using speed smart control system with changing the torque

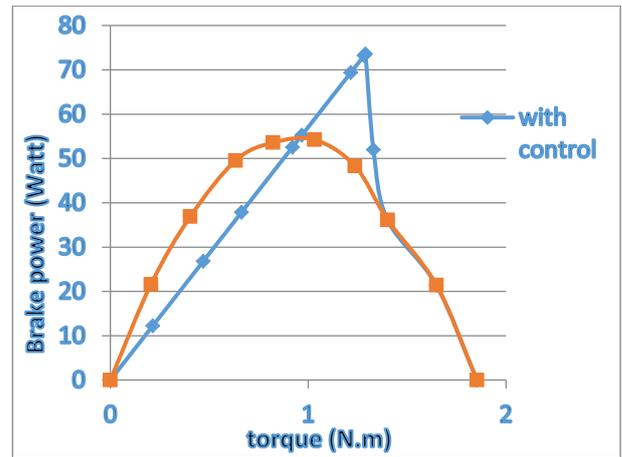


Figure8: Variation of brake power and torque of Pelton wheel with SSCS

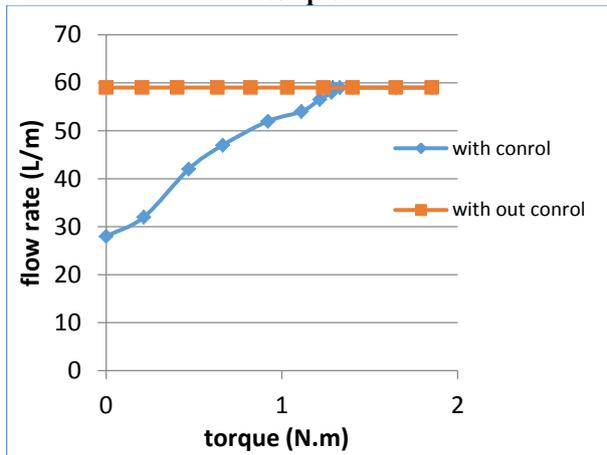


Figure 6: The relationship between the water flow rate and the torque of Pelton turbine with SSCS

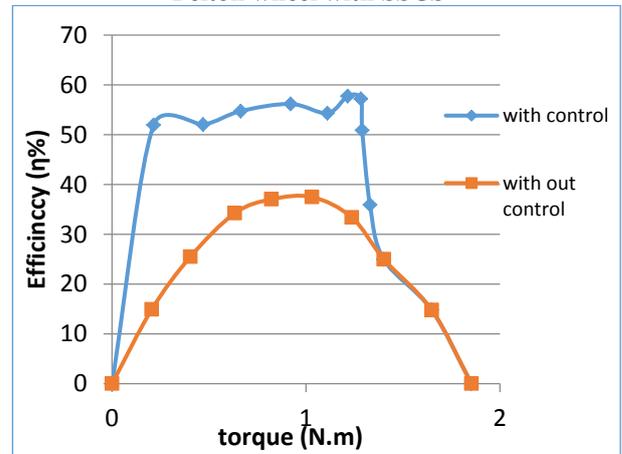


Figure 9: Variation of efficiency and the torque of Pelton turbine with SSCS

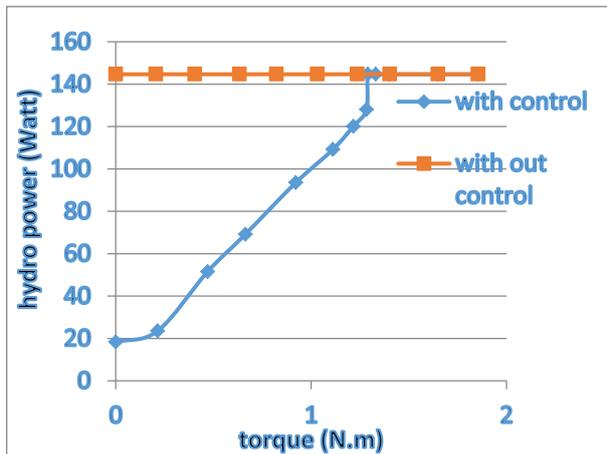


Figure 7: The relationship between the hydropower and the torque of Pelton turbine with SSCS

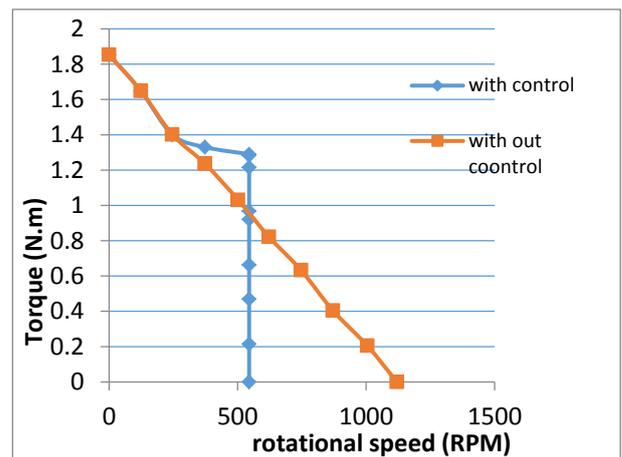


Figure 10: Variation of the torque and rotational speed of Pelton turbine with SSCS

Table 1: Performance of Pelton turbine with a speed smart control system

| No of reading | Efficiency (η %) | Ph [N.m/s] | Pb [N.m/s] | Ω [rad/s] | T [N.m] | W2 [Kg] | W1 [Kg] | N mean (rpm) | D [mm] | Q [l/min] | H [mH2O] |
|---------------|------------------------|------------|------------|------------------|---------|---------|---------|--------------|--------|-----------|----------|
| 1 | 0 | 18.321 | 0 | 57.0722 | 0 | 0 | 0 | 545 | 8.87 | 28 | 4 |
| 2 | 51.9355 | 23.544 | 12.22771 | 57.0722 | 0.21425 | 0.36 | 1.4 | 545 | 8.87 | 32 | 4.5 |
| 3 | 52.04 | 51.5025 | 26.806 | 57.0722 | 0.4697 | 0.62 | 2.9 | 545 | 8.87 | 42 | 7.5 |
| 4 | 54.74 | 69.16 | 37.858 | 57.0722 | 0.6633 | 0.78 | 4 | 545 | 8.87 | 47 | 9 |
| 5 | 56.18 | 93.522 | 52.555 | 57.0722 | 0.92086 | 1.13 | 5.6 | 545 | 8.87 | 52 | 11 |
| 6 | 49.33 | 112 | 55.2599 | 57.0722 | 0.9682 | 1.3 | 6 | 545 | 8.87 | 55 | 12.5 |
| 7 | 57.76 | 120.09 | 69.3689 | 57.0722 | 1.2154 | 1.6 | 7.5 | 545 | 8.87 | 56.5 | 13 |
| 8 | 57.19 | 128.02 | 73.223 | 57.0722 | 1.283 | 1.77 | 8 | 545 | 8.87 | 58 | 13.5 |
| 9 | 50.88 | 144.69 | 73.6 | 57.0722 | 1.29 | 2.5 | 8.8 | 545 | 8.87 | 59 | 15 |
| 10 | 35.91 | 144.69 | 51.96 | 39.07619 | 1.33 | 4 | 10.5 | 373 | 8.87 | 59 | 15 |
| 11 | 24.96 | 144.69 | 36.13043 | 25.77143 | 1.401 | 4.82 | 11.93 | 246 | 8.87 | 59 | 15 |
| 12 | 14.79 | 144.69 | 21.41393 | 12.99048 | 1.648 | 5.51 | 13.87 | 124 | 8.87 | 59 | 15 |
| 13 | 0 | 144.69 | 0 | 0 | 1.853 | 6.2 | 15.6 | 0 | 8.87 | 59 | 15 |

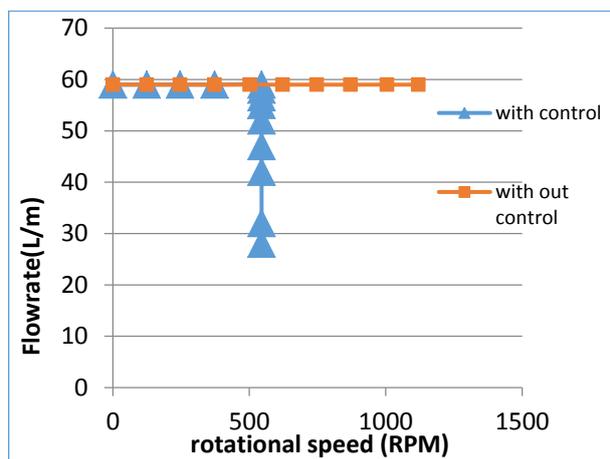


Figure 11: Variation of water flow rate and rotational speed of Pelton turbine with SSCS

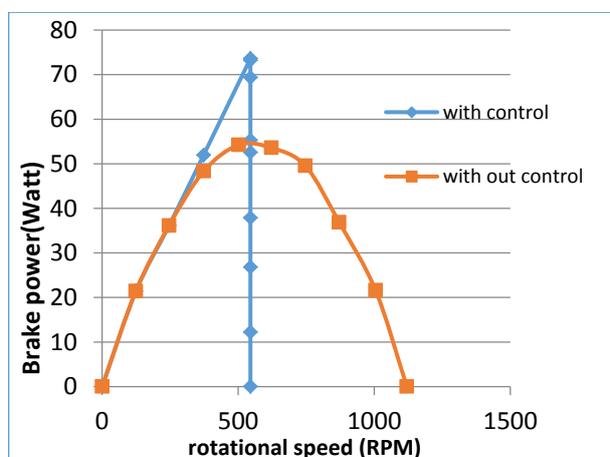


Figure 12: Variation of brake power and rotational speed of Pelton turbine with SSCS

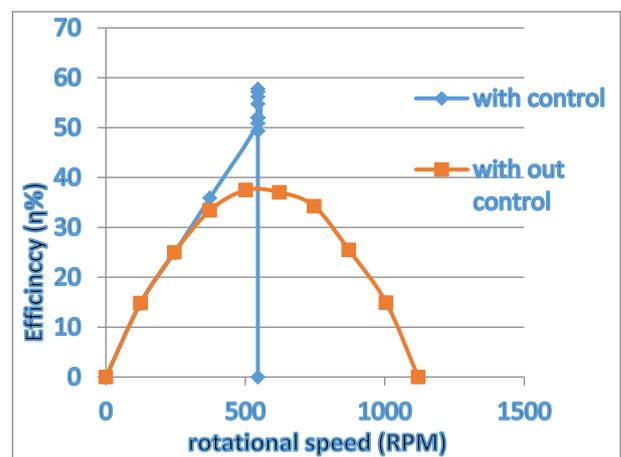


Figure 13: Variation of efficiency and rotational speed of Pelton turbine with SSCS

5. Conclusions

Based on the previous discussion of the obtained results, the following conclusion can be extracted.

1. The rotational speed time response about four seconds to achieve a steady state rotational speed of Pelton turbine.
2. It was observeda three regions (show in the figures) of performance characteristic for Pelton

turbine in the case of using Smart Speed Control System.

3. The use of Speed Smart Control System provides more stability to the Pelton turbine operation over wide range of water flow rate, so it makes the Pelton turbine work with high efficiency, power generated, and low amount of water consumed.

4. Using a Speed Smart Control System reduces a hydraulic power (input power) of the water according to the load applied to the shaft. The maximum reduction in the hydraulic power was approximately equate to 87.33% at zero torque.

5. The optimum torque for maximum brake power and efficiency of Pelton turbine system with smart speed control system is increased by 28.15%.

6. The Pelton turbine system performance with SSCS have been improved and enhanced. The percentage increase in the brake power and efficiency of Pelton turbine system were 26.3% and 35% respectively at the optimum torque for maximum brake power and efficiency of Pelton turbine system.

Nomenclature

| Abbreviation | Meaning of Abbreviation | Abbreviation | Manning of Abbreviation |
|--------------|----------------------------------|--------------|---------------------------------------|
| D | Nozzle diameter (m) | Q | Discharge (1/S) |
| F1 | Load (N) | R | Brake wheel radius 0.021m |
| F2 | Load (N) | SSCS | Speed smart control system |
| g | Acceleration m ² /S | t | Time (S) |
| H | Head of water m H ₂ O | T | Torque (N*m) |
| N | Revolution per minutes (rpm) | V | Volume of water |
| Ph | Input hydropower (W) | ω | Angular speed (rad/S) |
| Pb | Brake Power (W) | η | Efficiency |
| | | ρ | Density of water (kg/m ³) |

References

[1] Iraqi Meteorological Organization and Seismology, 2015, www.meteoseism.gov.iq.

[2] A. Lejeune, and S.L. hui, "Hydropower: A Multi Benefit Solution For Renewable Energy, Comprehensive Renewable Energy", Vol. 6, 15-47, 2012.

[3] Available at http://en.wikipedia.org/wiki/Pelton_wheel

[4] K.Y.VP. Niranjan, B.P. Hima, S.A. Divya, A. Sravani, "A Novel Implementation of Phase Control Technique for Speed Control of Induction Motor Using ARDUINO", International Journal of Emerging Technology and Advanced Engineering, Vol. 3 (4), pp. 469-473, 2013.

[5] R.P. Paul, G.B. Rathod, V.R. Trivedi, P.V. Thakkar, "Persistence of Vision Control Using Arduino", I.J. Intelligent Systems and Applications, Vol. 0, No. 1, 102-111, 2014.

[6] N. Rai, B. Rai, "Neural Network based Closed loop Speed Control of DC Motor using Arduino Uno", International Journal of Engineering Trends and Technology, Vol. 4, No. 2, 137-140, 2013.

[7] B. K.C. BholaThapa, "Pressure Distribution at Inner Surface of Selected Pelton Bucket for Micro Hydro", Kathmandu University Journal of Science, Engineering and Technology, Sept, Vol. 5, No. 2, 42-50, 2009.

[8] Z. Zhang, "Inlet flow conditions and the jet impact work in a Pelton turbine", Journal Power and Energy, Vol. 223, 589-569, 2009.

[9] M.K. Padhy, R.P. Saini, "Effect of size and concentration of silt particles on erosion of Pelton turbine buckets", Journal of Energy, Vol. 34, 1477-148, 2009.

[10] D. Jošt, MežnarP and A. Lipej, "Numerical prediction of Pelton turbine efficiency", IOP Conference Series Earth and Environmental Science, September 2010.

[11] L.E. Klemetsen, "An experimental and numerical study of the free surface Pelton bucket flow", Master Thesis, Norwegian University of Science and Technology, Norway, 2010.

[12] F.G. Stamatelos, J. S. Anagnostopoulos, and D.E. Papantonis", Performance measurements on a Pelton turbine model", Journal of Power and Energy, Vol. 225, No. 351, 351-362, 2011.

[13] B.W. Solimslie and O.G. Dahlhaug, "A reference Pelton turbine design", 6th, IAHR Symposium on Hydraulic Machinery and Systems, IOP Publishing, IOP Conf. Series. Earth and Environmental Science. 15, 2012.

[14] A. Rossetti, G. Pavesi, G. Cavazzini, A.Santolin and G.Ardizzon, "Influence of the bucket geometry on the Pelton performance", Journal Power and Energy, Vol. 228, No. 1, 33-45, 2014.

[15] Y. Xiao, Z. Wang, J. Zhang, C. Zeng and Z. Yan, "Numerical and experimental analysis of the hydraulic performance of a prototype Pelton turbine", Journal of Power and Energy, Vol. 228, No. 1, 46-55, 2014.

[16] J.L. Chukwuneke, C.H. Achebe, P.C. Okolie., H.A. Okwudibe, "Experimental Investigation on Effect of Head and Bucket Splitter Angle on the Power Output of a Pelton Turbine", International Journal of Energy Engineering, 4, 4, 81-87, 2014.

[17] B Vinod, B Biksham, V. Janeyulu, "Design and Analysis Of Pelton Wheel", International Journal & Magazine Of Engineering, Technology, Management and Research, Vol. 1, No. 12, 549-558, 2014.

[18] J. L. Chukwuneke, C.H. Achebe, M.C. Nwosu, J.E. Sinebe, "Analysis And Simulation On Effect Of Head And Bucket Splitter Angle On The Power Output Of A Pelton Turbine", International Journal of

Engineering and Applied Sciences, Vol. 5, 3, 1-8, 2014.

[19] D. Bisen, S.K. Shukla, P.K. Sharma, "Review Paper on Nozzle in Hydro-Turbine", International Journal of Advanced Technology in Engineering and Science, Vol. 2, No. 8, 481-487, 2014.

[20] K. Shimokawa, A. Furukawa, K. Okuma, D. Matsushita, S. Watanabe, "Experimental study on simplification of Darrieus-type hydro turbine with inlet nozzle for extra-low head hydropower utilization", Renewable Energy, Vol. 41, 376-382, 2012.

[21] S.A. John, D. PAPANTONIS, "A fast Lagrangian simulation method for flow analysis and runner design in Pelton turbines", Journal of Hydrodynamics, Ser. B, Vol. 24, Issue 6, 930-941, 2012.

[22] J.F Wang, J. Piechna, N. Müller, "A novel design of composite water turbine using CFD", Journal of Hydrodynamics, Ser. B, Vol. 24, Issue 1, 11-16, 2012.

[23] A.A. Khan, A.M. Khan, M. Zahid, R. Rizwan, "Flow acceleration by converging nozzles for power generation in existing canal system", Renewable Energy, Vol. 60, 548-552, 2013.

[24] M. Massini, H. Yang, J.C. Han, "The benefit of high-conductivity materials in film cooled turbine nozzles", International Journal of Heat and Fluid Flow, Vol. 34, 107-116, 2012.

[25] F.P.F. Yu, B.T. Zhang, H.C. Zhou, "The influence of back pressure on the flow discharge coefficients of plain orifice nozzle", International Journal of Heat and Fluid Flow, Vol. 44, 509- 514, 2013.

[26] Z. Lan, D. Zhu, W. Tian, G. Su, S. Qiu, "Experimental study on spray characteristics of pressure-swirl nozzles in pressurizer," Annals of Nuclear Energy, Vol. 63, 215-227, 2014.

[27] J.L. Xie, Z.W. Gan, F. Duan, "Characterization of spray atomization and heat transfer of pressure swirl nozzles", International Journal of Thermal Sciences, Vol. 68, 94-102, 2013.

[28] T.R. Bajracharya, B. Acharya, C.B. Joshi, R.P. Saini, O.G. Dahlhaug, "Sand erosion of Pelton turbine nozzles and buckets: A case study of Chilime Hydropower Plant", Wear, Vol. 264, Issues 3-4, 4, 177-184, 2008.

[29] A. Date, A. Akbarzadeh, "Design and analysis of a split reaction water turbine", Renewable Energy, Vol. 35, Issue 9, 1947-1955, 2010.

Author(s) biography

T.Z. Farge



Received the B.Sc. in Mechanical Eng., al Rasheed Collage of Engineering and Science from University of Technology, Baghdad 1982, MSc. and Ph.D. degrees from University of Liverpool, United Kingdom, in

1982, and 1989 respectively, all in Mechanical Engineering. He is currently Lecturer Electromechanical Engineering department in University of Technology, Baghdad, Iraq. His research interests include fluid mechanics, heat transfer, hydraulic systems, renewable energy, and thermodynamics.

A.J. Owai



Received the B.Sc., MSc. and Ph.D. degrees from University of Technology, Baghdad, in 1996, 2000, and 2009 respectively, all in Electrical Engineering. He is currently Lecturer in Electromechanical Engineering department in University of Technology, Baghdad, Iraq. His research interests include electrical machines, special machines, power electronics, resonant converters, soft-switching techniques.

M. A. Qasim



Born in Baghdad (1985). Earned his BSc in Electrical Engineering from University of Technology (2006), MSc degrees in Electro Mechanical System Engineering from University of Technology (2016), Baghdad, Iraq. Work in Ministry of Health since 2009 as an electrical engineer deals with hospitals Projects.